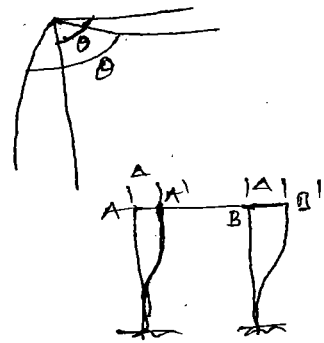


SLOPE DEFLECTION METHOD

- Using this method basic unknowns like slopes and deflections of joints can be calculated.
- Moments at the ends of a member is first written in terms of unknown slopes and deflections of the end joints.
- Considering joint equilibrium conditions, a set of equations are formed and solution of these equations gives unknown slopes and deflections.
- Then end moments of individual members are determined.
- Since it involves solution of simultaneous equations, a problem with more than three unknowns is considered as difficult for hand calculation.
- Development of this method in the matrix form, has led to the "Stiffness matrix method".

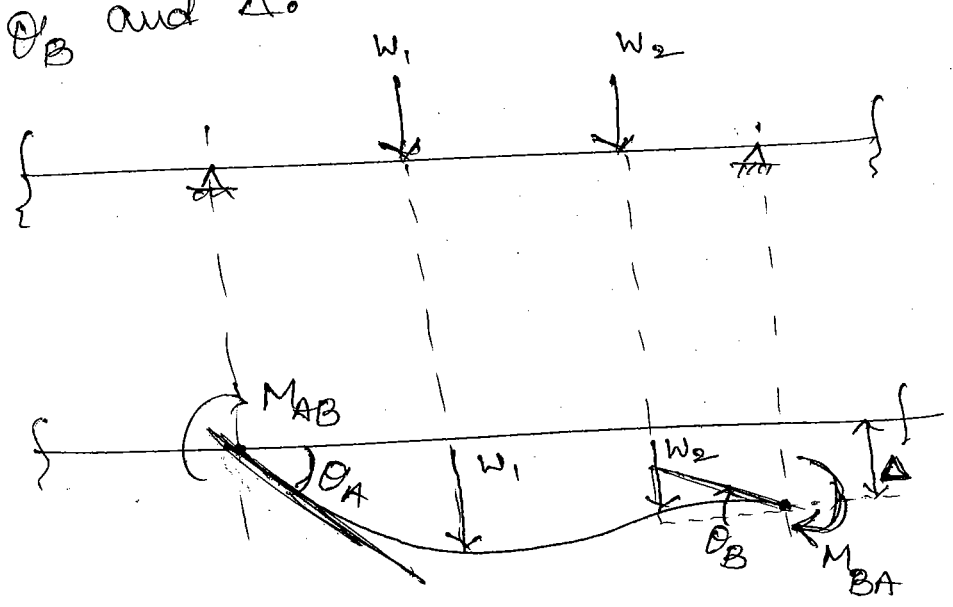
- Assumptions -
1. All joints are rigid,
 2. Distortions due to axial deformations are neglected
 $AA' = BB' = \Delta$
 3. Shear deformations are neglected.



Sign Conventions:-

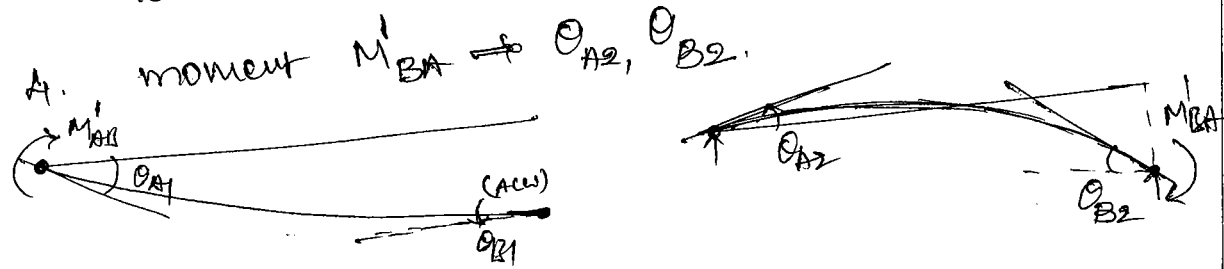
1. C.W moments are +ve; A.C.W moments are -ve
2. C.W Rotations " +ve; A.C.W Rotations " -ve
3. Settlement Δ is +ve if Right hand side support settles down
" " -ve " left " " "

Let AB , shown below, be a member of a rigid structure. After loading it undergoes deformations. Final moments at end A and end B are M_{AB} and M_{BA} . Now our aim is to derive the relationship between these final end moments and their displacements θ_A , θ_B and Δ .



The solution & development of final moments can be visualised in the following stages.

1. Due to given loadings end moments M_{FAB} , M_{FBA} develop without any rotations at ends. Hence called fixed end moments.
2. Settlement Δ takes place without any rotations at ends. This is similar to the settlement of supports in a fixed beam. Hence moments developed are $\frac{6EI\Delta}{L^2}$
3. moment M'_{AB} comes into play in SSB to cause end rotations θ_{A1} , θ_{B1} at A & B resp.



$$\theta_{A1} = \frac{M'_{AB}L}{3EI}$$

$$\theta_{B1} = \frac{M'_{BA}L}{6EI}$$

$$\theta_{A2} = \frac{M'_{BA}L}{6EI}$$

$$\theta_{B2} = \frac{M'_{AB}L}{3EI}$$

$$\theta_A = \theta_{A1} - \theta_{A2} = \frac{M'_{AB}L}{3EI} - \frac{M'_{BA}L}{6EI}$$

$$\theta_B = \theta_{B2} - \theta_{B1} = \frac{M'_{BA}L}{3EI} - \frac{M'_{AB}L}{6EI}$$

$$2\theta_A + \theta_B = \frac{2M'_{AB}L}{3EI} - \frac{M'_{BA}L}{3EI} + \frac{M'_{BA}L}{3EI} - \frac{M'_{AB}L}{6EI}$$

$$= \frac{3M'_{AB}L}{6EI} \Rightarrow M'_{AB} = \frac{2EI}{L} (2\theta_A + \theta_B)$$

$$\text{Similarly } M'_{BA} = \frac{2EI}{L} (2\theta_B + \theta_A)$$

due to Support Settlement

$$\therefore M_{AB} = M_{FAB} - \frac{6EI\Delta}{L^2} + M'_{AB} = M_{FAB} + M'_{AB} + M'_{AB}$$

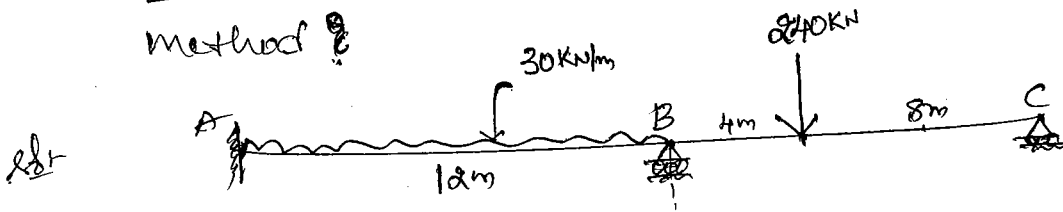
$$= M_{FAB} + \frac{2EI}{L} (2\theta_A + \theta_B) - \frac{6EI\Delta}{L^2}$$

$$M_{AB} = M_{FAB} + \frac{2EI}{L} \left[(2\theta_A + \theta_B) - \frac{3\Delta}{L} \right] \quad \text{--- (1)}$$

$$M_{BA} = M_{FBA} + \frac{2EI}{L} \left[(2\theta_B + \theta_A) - \frac{3\Delta}{L} \right] \quad \text{--- (2)}$$

① & ② Are slope deflection Equations

over rollers at B and C. $AB = BC = 12\text{m}$. Beam carries a UDL of 30kN/m over AB and a point load of 240kN at a distance 4m from B on span BC. B has a settlement of 30mm . $E = 2 \times 10^5 \text{ N/mm}^2$, $I = 2 \times 10^9 \text{ mm}^4$. Analyse the beam by slope deflection method.



writing FEMs

$$M_{FAB} = -\frac{30 \times 12^2}{12} = -360 \text{ kNm}$$

$$M_{FBA} = 360 \text{ kNm}$$

$$M_{FBC} = -\frac{240 \times 4 \times 8}{12^2} = -426.67 \text{ kNm}$$

$$M_{FCB} = +\frac{240 \times 8 \times 4}{12^2} = 213.33 \text{ kNm}$$

$$\therefore M_{AB} = M_{FAB} + \frac{2EI}{L} \left(2\theta_A + \theta_B - \frac{3\Delta}{L} \right)$$

$$= -360 + \frac{2 \times 2 \times 10^5}{12} \left[2\theta_A + \theta_B - \frac{3 \times \left(\frac{30}{1000} \right)}{12} \right]$$

$$= -360 + \frac{2 \times 10^5}{3} \left(2\theta_A + \theta_B - \frac{3}{400} \right)$$

$$M_{BA} = 360 + \frac{2 \times 2 \times 10^5}{12} \left(2\theta_B + \theta_A - \frac{3 \times \left(\frac{30}{1000} \right)}{12} \right)$$

$$= 360 + \frac{2 \times 10^5}{3} \left(\theta_A + 2\theta_B - \frac{3}{400} \right)$$

$$M_{BC} = -426.67 + \frac{2 \times 2 \times 10^5}{12} \left(2\theta_B + \theta_C + \frac{3 \times \left(\frac{30}{1000} \right)}{12} \right)$$

$$= -426.67 + \frac{2 \times 10^5}{3} \left(2\theta_B + \theta_C + \frac{3}{400} \right)$$

$$M_{CB} = 213.33 + \frac{2 \times 10^5}{3} \left(\theta_B + 2\theta_C + \frac{3}{400} \right)$$

Substituting, $\theta_A = 0$ and $\sum M_B = 0$,

$$M_{BA} + M_{BC} = 0 \rightarrow$$

$$360 + \frac{2 \times 10^5}{3} \left[\theta_A + 2\theta_B - \frac{3}{400} \right] + (-426.67) +$$

$$\frac{2 \times 10^5}{3} \left[2\theta_B + \theta_C + \frac{3}{400} \right] = 0$$

$$4\theta_B + \theta_C = \frac{66.67 \times 3}{2 \times 10^5} \rightarrow \textcircled{1}$$

$$M_C = 0 \Rightarrow M_{CB} = 0 \Rightarrow 213.33 + \frac{2 \times 10^5}{3} (\theta_B + 2\theta_C + \frac{3}{400}) = 0$$

$$\frac{2 \times 10^5}{3} (\theta_B + 2\theta_C) = -713.33$$

$$-\theta_B + 2\theta_C = + \frac{713.33 \times 3}{2 \times 10^5}$$

$$2(4\theta_B) + 2\theta_C = \frac{66.67 \times 6}{2 \times 10^5}$$

$$7\theta_B = \frac{2540}{2 \times 10^5} \Rightarrow \boxed{1.814 \times 10^{-3} = \theta_B}$$

$$\theta_C = \frac{66.667 \times 3}{2 \times 10^5} - 4 \times 1.814 \times 10^{-3}$$

$$\boxed{\theta_C = -6.256 \times 10^{-3}}$$

$$\therefore M_{AB} = -360 + \frac{2 \times 10^5}{3} \left[(0 + 1.814 \times 10^{-3}) - \frac{3}{400} \right] = \underline{\underline{-739.067 \text{ kNm}}}$$

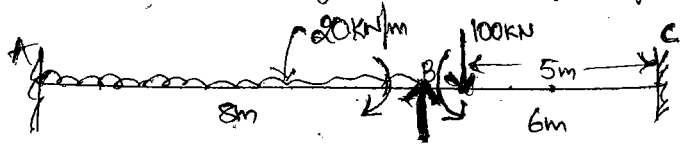
$$M_{BA} = 360 + \frac{2 \times 10^5}{3} \left[0 + 2 \times 1.814 \times 10^{-3} - \frac{3}{400} \right] = \underline{\underline{101.867 \text{ kNm}}}$$

$$M_{BC} = -426.67 + \frac{2 \times 10^5}{3} \left[2 \times 1.814 \times 10^{-3} + (-6.256 \times 10^{-3}) + 7.5 \times 10^{-3} \right] = \underline{\underline{-101.867 \text{ kNm}}}$$

$$\underline{\underline{M_{CB} = 0}}$$

Set 2, 3, & 4 \Rightarrow Similar problems with data change

Diagrams of beam in below figure by slope deflection method?



Ans

writing FEMM $M_{FAB} = -\frac{20 \times 8^2}{12} = -106.67 \text{ KNm}$

$M_{FBA} = \frac{20 \times 8^2}{12} = 106.67 \text{ KNm}$

$M_{FBC} = -\frac{100 \times 1 \times 5^2}{6^2} = -69.444 \text{ KNm}$

$M_{FCB} = \frac{100 \times 1^2 \times 5}{6^2} = 13.889 \text{ KNm}$

Slope-Deflection Equations

$M_{AB} = -106.67 + \frac{2EI}{8} (\theta_A + \theta_B - 0)$

$M_{BA} = 106.67 + \frac{2EI}{8} (\theta_A + 2\theta_B - 0)$

$M_{BC} = -69.444 + \frac{2EI}{6} (2\theta_B + \theta_C - 0)$

$M_{CB} = 13.889 + \frac{2EI}{6} (\theta_B + 2\theta_C - 0)$

Substituting, $\theta_A = 0$; $\theta_C = 0$

Joint Equilibrium equation, $\sum M_B = 0$; $M_{BA} + M_{BC} = 0$

$106.67 + \frac{2EI}{8} (0 + 2\theta_B) - 69.444 + \frac{2EI}{6} (2\theta_B + 0) = 0$

$37.226 + \frac{4EI\theta_B}{8} + \frac{4EI\theta_B}{6} = 0$

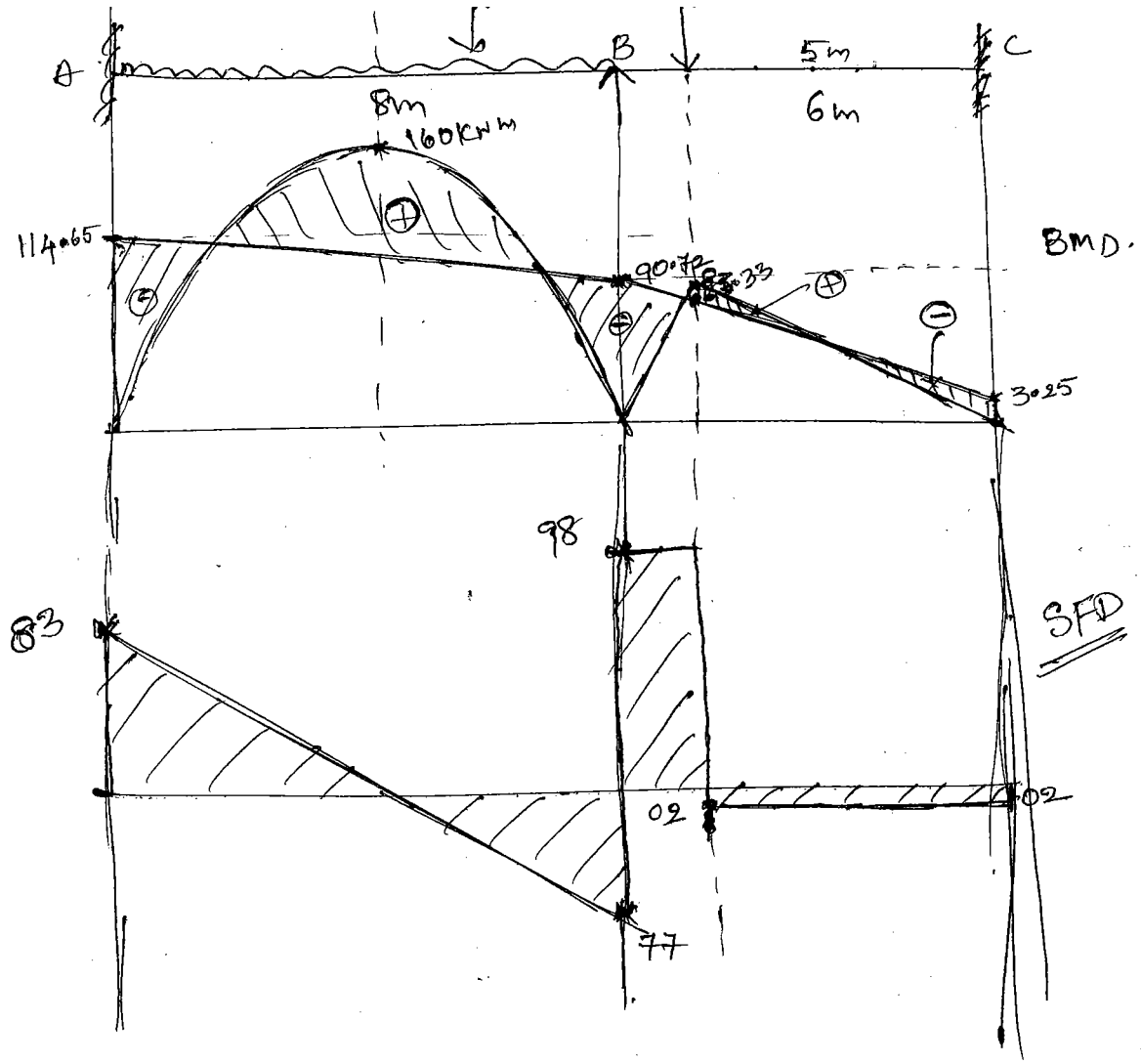
$EI\theta_B (\frac{1}{2} + \frac{2}{3}) = -37.226$ $\theta_B = -\frac{31.908}{EI}$

$\therefore M_{AB} = -106.67 + \frac{2EI}{8} (0 - \frac{31.908}{EI}) = -114.647 \text{ KNm}$

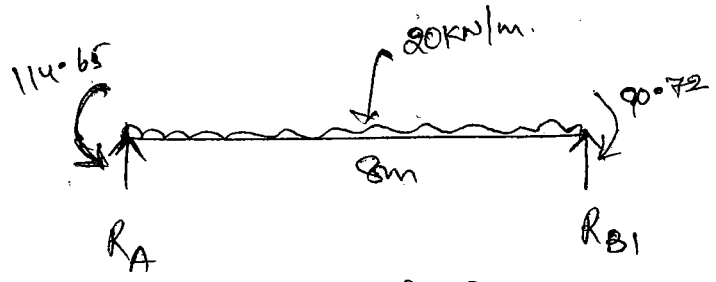
$M_{BA} = 106.67 + \frac{2EI}{8} (0 - 2 \times \frac{31.908}{EI}) = 90.716 \text{ KNm}$

$M_{BC} = -69.444 + \frac{2EI}{6} (2 \times -\frac{31.908}{EI}) = -90.716 \text{ KNm}$

$M_{CB} = 13.889 + \frac{2EI}{6} (-\frac{31.908}{EI}) = 3.253 \text{ KNm}$



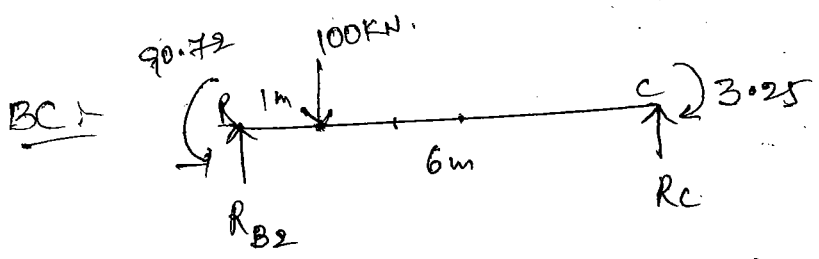
Shear force AD



$\sum M_A = 0$

$$114.65 + R_{B1} \times 8 - 90.72 - 20 \times 8 \times 4 = 0 \Rightarrow R_{B1} = 77 \text{ kN}$$

$$R_A = 83 \text{ kN}$$

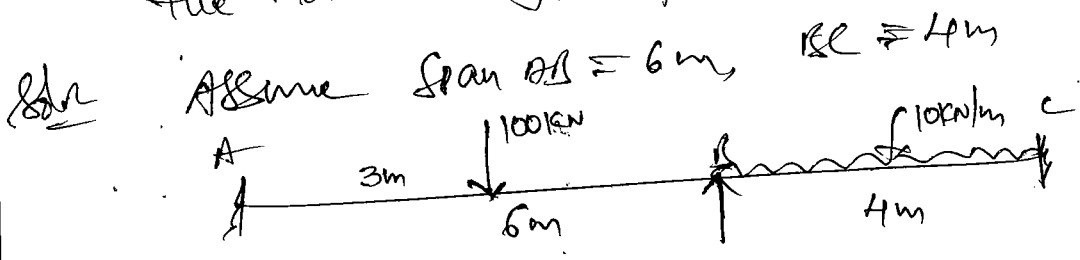


$\sum M_B = 0$

$$90.72 + R_C \times 6 - 3.25 - 100 = 0 \Rightarrow R_C = 2.0 \text{ kN}$$

$$R_{B2} = 98 \text{ kN}$$

EI throughout its length. The end supports A and C are fixed and beam is continuous over middle support B. Span BC is uniformly loaded with 10 kN/m length while a concentrated vertical load of 100 kN acts at the mid span of AB. Calculate the moments by slope deflection method?



FEM $M_{FAB} = -\frac{100 \times 6}{8} = -75$
 $M_{FBA} = 75$
 $M_{FBC} = -\frac{10 \times 4^2}{12} = -13.333$
 $M_{FCB} = 13.333$

SDEM

$$M_{AB} = -75 + \frac{2EI}{6}(\theta_A + \theta_B) = -75 + \frac{2EI}{6}(\theta_B)$$

$$M_{BA} = 75 + \frac{2EI}{6}(\theta_A + 2\theta_B) = 75 + \frac{2EI}{6}(2\theta_B)$$

$$M_{BC} = -13.333 + \frac{2EI}{4}(2\theta_B + \theta_C) = -13.333 + \frac{2EI}{4}(2\theta_B)$$

$$M_{CB} = 13.333 + \frac{2EI}{4}(\theta_B + 2\theta_C) = 13.333 + \frac{2EI}{4}(\theta_B)$$

Joint Eqn $\sum M_B = 0 ; M_{BA} + M_{BC} = 0$

$$75 + \frac{2EI}{6}(2\theta_B) - 13.333 + \frac{2EI}{4}(2\theta_B) = 0$$

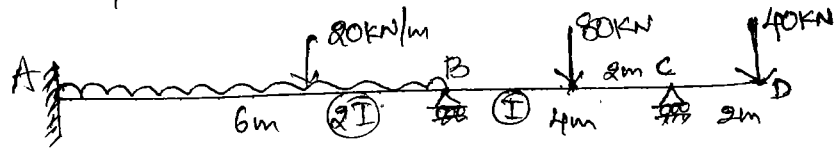
$$EI \theta_B \left(\frac{2}{3} + 1\right) = -61.667$$

$$EI \theta_B = \frac{-61.667 \times 3}{5} = -37$$

$\theta_B = \frac{-37}{EI}$

$M_{AB} = -75 + \frac{2EI}{6} \left(\frac{-37}{EI}\right) = -87.333 \text{ kNm}$
 $M_{BA} = 75 + \frac{2EI}{6} \left(\frac{-37}{EI}\right) = 50.333 \text{ kNm}$
 $M_{BC} = -13.333 + \frac{2EI}{4} \left(\frac{-37}{EI}\right) = -50.333 \text{ kNm}$
 $M_{CB} = 13.333 + \frac{2EI}{4} \left(\frac{-37}{EI}\right) = -5.667 \text{ kNm}$

by slope deflection method and draw BMD?



soln

Fixed end moments:- $M_{FAB} = -\frac{20 \times 6^2}{8} = -60 \text{ kNm}$

$M_{FBA} = 60 \text{ kNm}$

$M_{FBC} = -\frac{80 \times 4}{8} = -40 \text{ kNm}$

$M_{FCB} = 40 \text{ kNm}$

Slope deflection Equations:- $M_{AB} = -60 + \frac{2EI(\theta_A + \theta_B - \psi)}{6}$

$M_{BA} = 60 + \frac{2EI(\theta_A + \theta_B - \psi)}{6} = -60 + \frac{2EI\theta_B}{3}$ ($\because \theta_A = 0$)

$= 60 + \frac{4}{3}EI\theta_B$

$M_{BC} = -40 + \frac{2EI(2\theta_B + \theta_C - \psi)}{4} = -40 + EI\theta_B + 0.5EI\theta_C$

$M_{CB} = 40 + \frac{2EI(\theta_B + 2\theta_C - \psi)}{4} = 40 + 0.5EI\theta_B + EI\theta_C$

Equilibrium Equations: ① $\sum M_B = 0$; $M_{BA} + M_{BC} = 0$

$60 + \frac{4}{3}EI\theta_B + (-40) + EI\theta_B + 0.5EI\theta_C = 0$

$2.333EI\theta_B + 0.5EI\theta_C = -20 \rightarrow$ ②

② $\sum M_C = 0 \Rightarrow M_{CB} + M_{CD} = 0$

But, $M_{CD} = -40 \times 2 = -80 \therefore M_{CB} = 80 \text{ kNm}$

$40 + 0.5EI\theta_B + EI\theta_C = 80$

$0.5EI\theta_B + EI\theta_C = 40$ — ③

Solving ② & ③ $EI\theta_B = 19.2$

$EI\theta_C = 49.6$

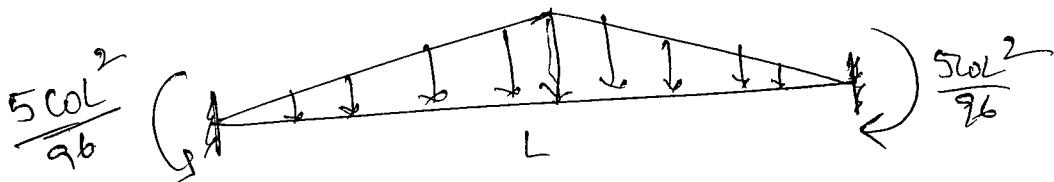
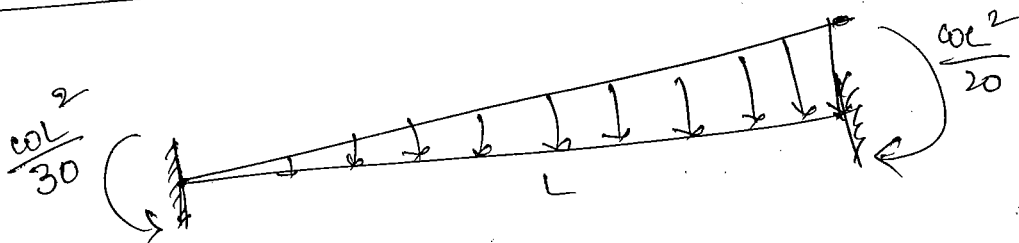
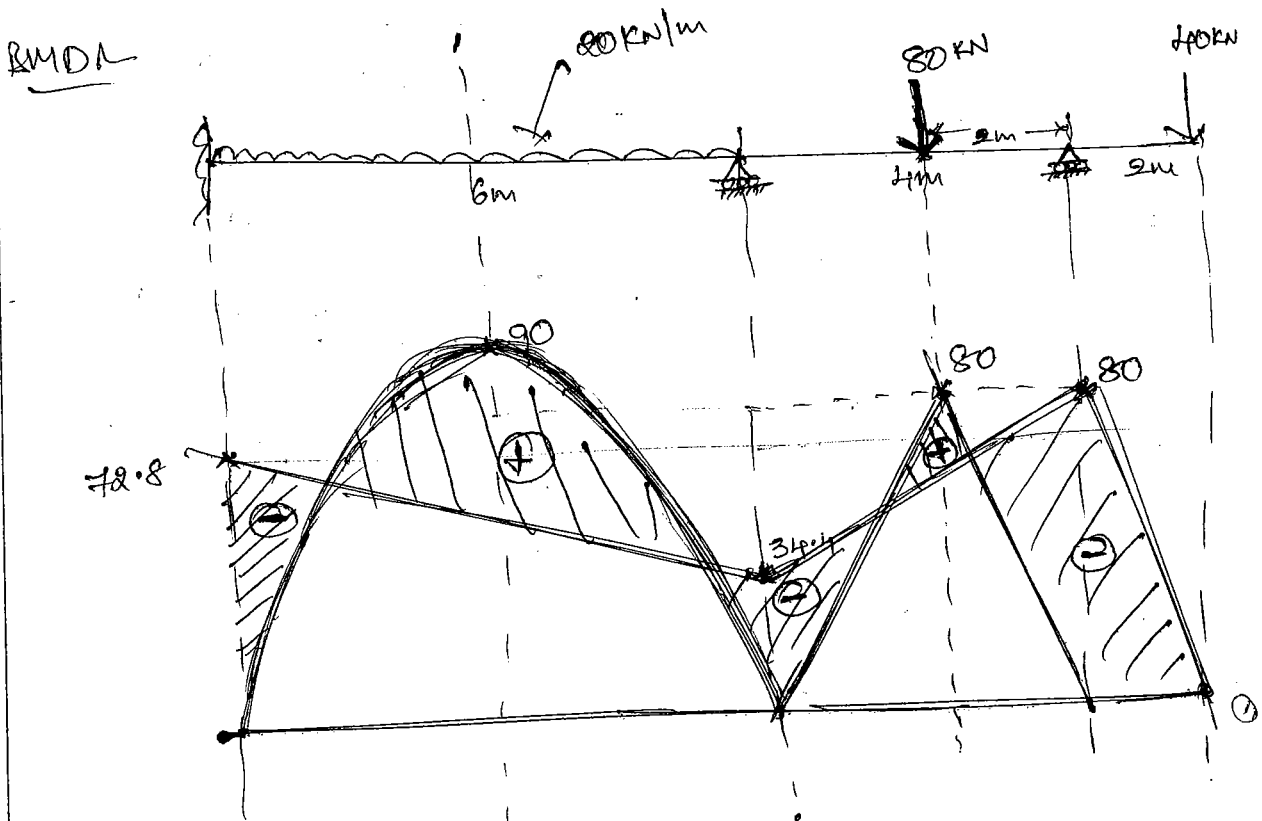
Substituting θ_B & θ_C in slope deflection equations we get fixed end moments

$$M_{AB} = -60 + \frac{8}{3}(-19.2) = -72.8 \text{ KNM}$$

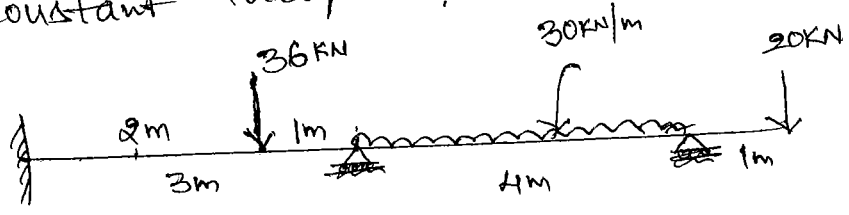
$$M_{BA} = 60 + \frac{4}{3}(-19.2) = 34.4 \text{ KNM}$$

$$M_{BC} = -40 + (-19.2) + 0.5(49.6) = -34.4 \text{ KNM}$$

$$M_{CB} = 80 \text{ KNM}$$

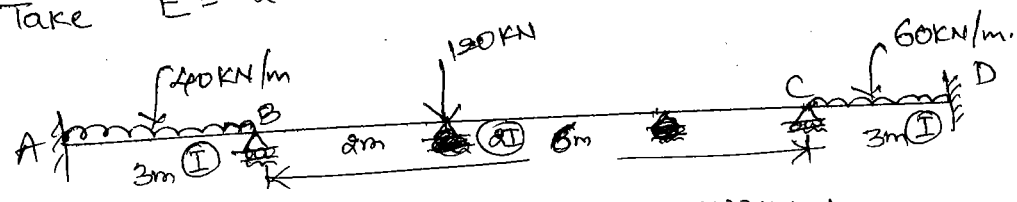


① Analyse the continuous beam shown in fig below by slope deflection method and draw bending moment diagram. flexural rigidity is constant throughout?



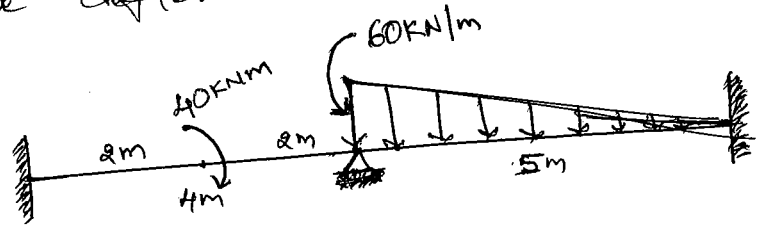
Ans: $M_{AB} = 2.88 \text{ kNm}$; $M_{BA} = 37.76 \text{ kNm}$; $M_{BC} = -M_{CB} = 20 \text{ kNm}$.

② Analyse the continuous beam ABCD shown in figure given below, if support C sink by 10mm. Take $E = 2 \times 10^5 \text{ N/mm}^2$ and $I = 4 \times 10^7 \text{ mm}^4$.

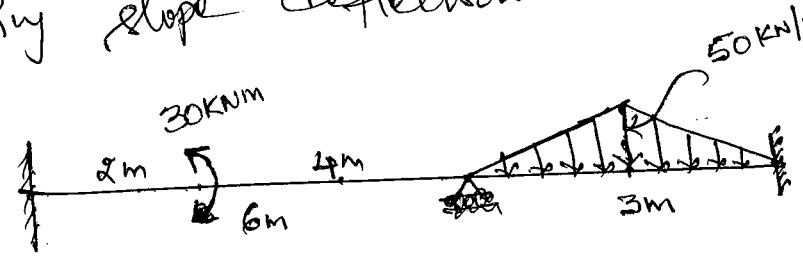


Ans:- $M_{AB} = -0.111 \text{ kNm}$; $M_{BA} = -M_{BC} = 89.777 \text{ kNm}$;
 $M_{CB} = -M_{CD} = 22.111 \text{ kNm}$; $M_{DC} = 22.111 \text{ kNm}$.

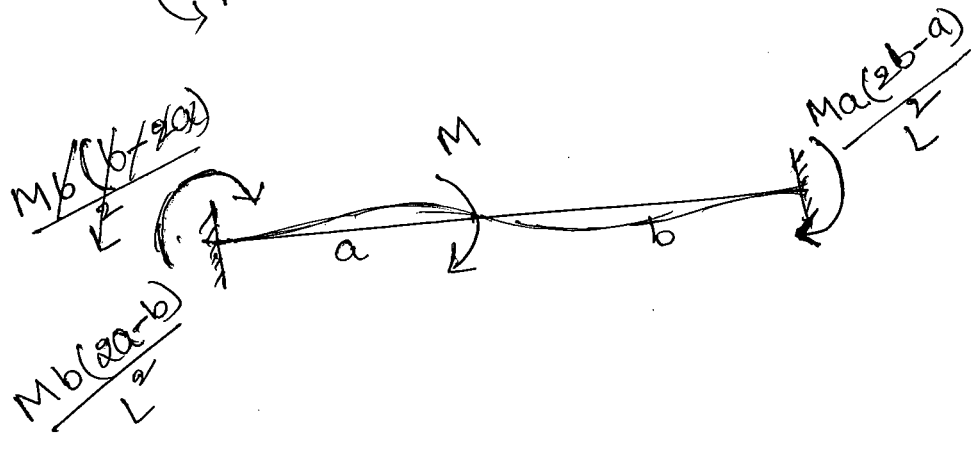
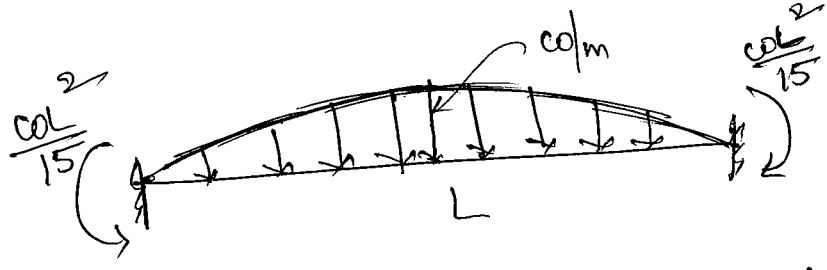
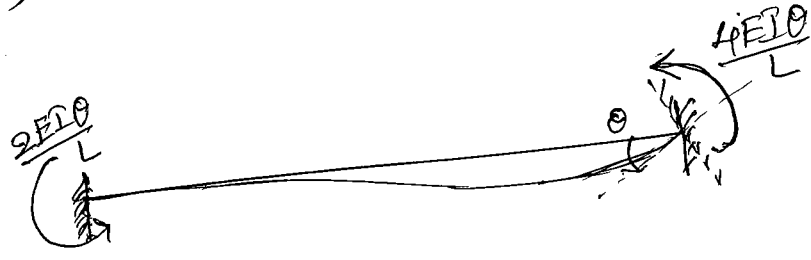
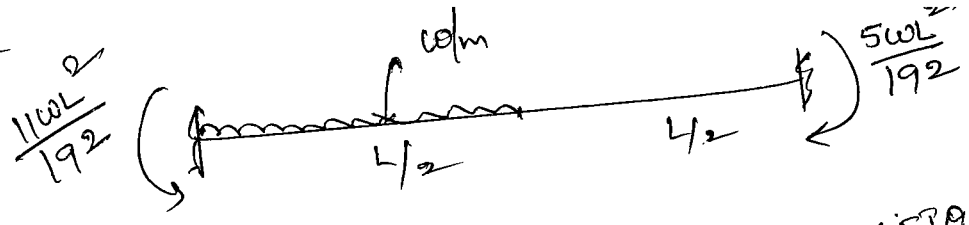
③ Analyse the continuous beam shown below using slope deflection method?



④ Analyse the continuous beam shown below using slope deflection method?

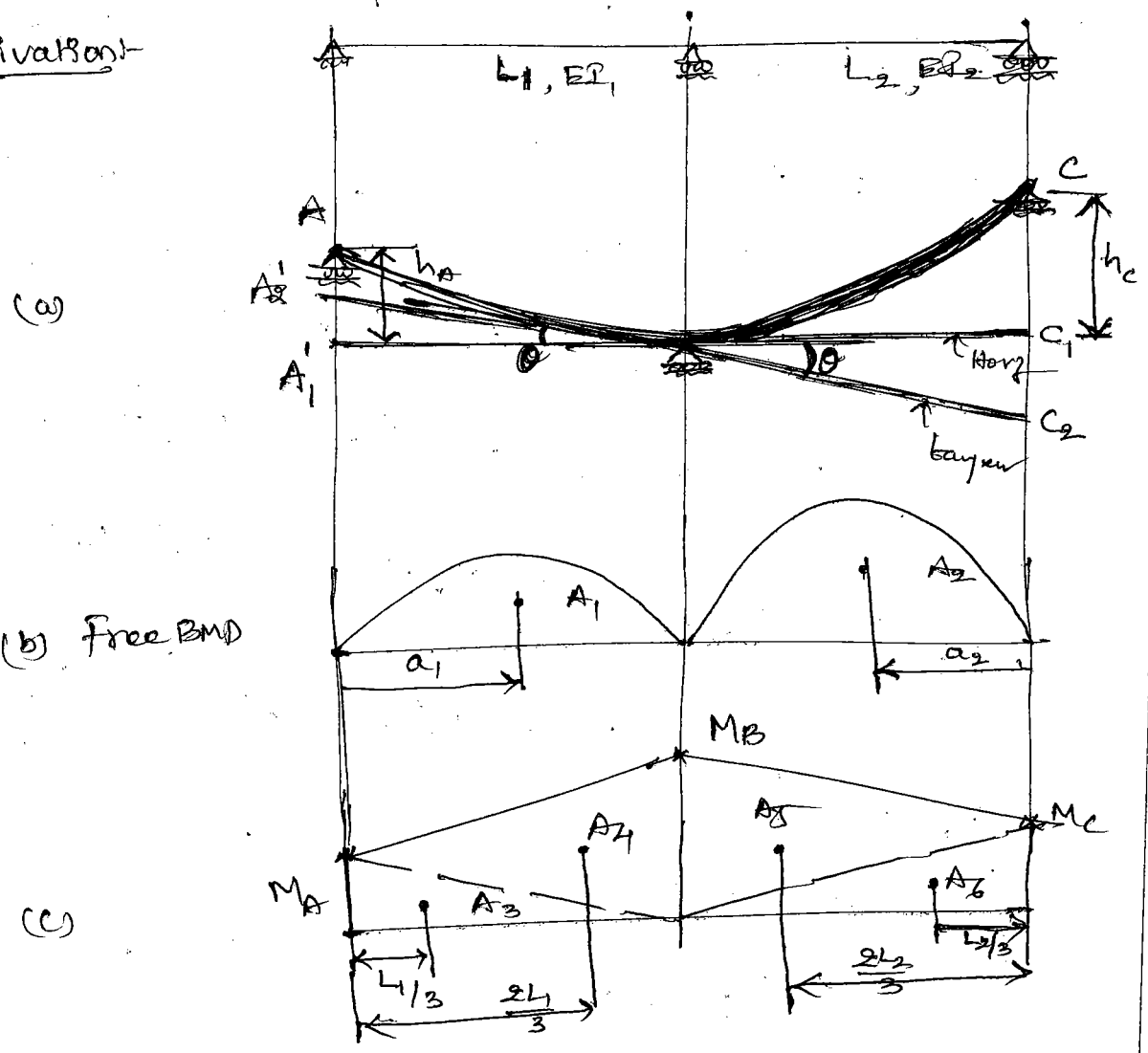


FEMs 1



CLAPEYRON'S Theorem:- It is the relation between the moments at three successive supports. This is derived from the consistency condition that the slope at the middle support calculated from left span and the right span should be same.

Derivation:-



From (a) $\tan \theta = \frac{A_1 A_2'}{L_1} = \frac{C_1 C_2}{L_2}$ ①

But $A_1 A_2' = AA_1' - AA_2' = h_A - AA_2'$
 $= h_A - \text{Vertical intercept of A from tangent at B}$
 $= h_A - \text{moment of } (M/EI) \text{ diagram w.r.t. B about A.}$

$$= h_A - \frac{1}{EI_1} \left[A_1 a_1 + \frac{A_3 L_1}{3} + A_4 \cdot \frac{2L_1}{3} \right]$$

$$= h_A - \frac{1}{EI_1} \left[A_1 a_1 + \left(\frac{1}{2} \times M_A \times L_1 \right) \frac{L_1}{3} + \frac{1}{2} \times M_B \times L_1 \times \frac{2L_1}{3} \right]$$

$$= h_A - \frac{1}{EI_1} \left[A_1 a_1 + \frac{M_A L_1^2}{6} + \frac{M_B L_1^2}{3} \right]$$

$$= h_A - \frac{1}{6EI_1} \left[6A_1 a_1 + M_A L_1^2 + 2M_B L_1^2 \right] \quad \text{--- (2)}$$

lly. $C_1 C_2 = CC_2 - CC_1$

= vertical intercept of \bar{C} from tangent at B

= moment of $\left(\frac{M}{EI} \right)$ w.r. to B about C - hc

$$= \frac{1}{EI_2} \left[A_2 a_2 + A_5 \cdot \frac{2L_2}{3} + A_6 \cdot \frac{L_2}{3} \right] - hc$$

$$= \frac{1}{EI_2} \left[A_2 a_2 + \frac{1}{2} \times M_B \times L_2 \times \frac{2L_2}{3} + \frac{1}{2} \times M_C \times L_2 \times \frac{L_2}{3} \right] - hc$$

$$= \frac{1}{6EI_2} \left[6A_2 a_2 + 2M_B L_2^2 + M_C L_2^2 \right] - hc \quad \text{--- (3)}$$

Substituting (2) & (3) in (1)

$$\frac{h_A}{L_1} - \frac{1}{6EI_1 L_1} \left[6A_1 a_1 + M_A L_1^2 + 2M_B L_1^2 \right] =$$

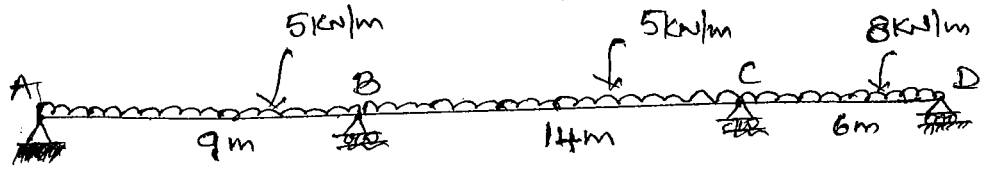
$$\frac{1}{6EI_2 L_2} \left[6A_2 a_2 + 2M_B L_2^2 + M_C L_2^2 \right] - \frac{hc}{L_2} \quad \times 6E$$

$$\frac{6Eh_A}{L_1} - \frac{1}{L_1} \left[6A_1 a_1 + M_A L_1^2 + 2M_B L_1^2 \right] = \frac{1}{L_2} \left[6A_2 a_2 + 2M_B L_2^2 + M_C L_2^2 \right] - \frac{6Ehc}{L_2}$$

$$M_A \left(\frac{L_1}{I_1} \right) + 2M_B \left(\frac{L_1}{I_1} + \frac{L_2}{I_2} \right) + M_C \left(\frac{L_2}{I_2} \right) = \frac{-6A_1 a_1}{I_1 L_1} - \frac{6A_2 a_2}{I_2 L_2}$$

$$+ \frac{6Eh_A}{L_1} + \frac{6Ehc}{L_2} \quad //$$

Prob:- A continuous beam ABCD is simply supported over three spans. Span AB is 9m carrying an UDL of 5kN/m. Span BC is 14m carrying an UDL of 5kN/m and span CD is 6m carrying an UDL of 8kN/m. Find the moment over supports B and C. Draw BMD?



Sol:- Applying three moment theorem for A, B & C supports

$$M_A \left(\frac{L_1}{I_1} \right) + 2M_B \left(\frac{L_1}{I_1} + \frac{L_2}{I_2} \right) + M_C \left(\frac{L_2}{I_2} \right) = -\frac{6A_1a_1}{I_1 L_1} - \frac{6A_2a_2}{I_2 L_2} + \frac{6Eh_A}{L_1} + \frac{6Eh_C}{L_2}$$

∵ $h_A = h_C = 0$

∵ EI is same throughout $I_1 = I_2 = I$

$$\therefore M_A L_1 + 2M_B (L_1 + L_2) + M_C L_2 = -\frac{6A_1a_1}{L_1} - \frac{6A_2a_2}{L_2}$$

$$M_A(9) + 2M_B(9+14) + M_C(14) = -\left(\frac{6 \times \frac{2}{3} \times 9 \times 5 \times 9^2}{8 \times 9} \times 4 \right) - \left(\frac{6 \times \frac{2}{3} \times 14 \times 5 \times 14^2}{8 \times 14} \times 7 \right) / 14$$

$$46M_B + 14M_C = \frac{-8201.25}{9} - \frac{48020}{14}$$

$$46M_B + 14M_C = -911.25 - 3430 = -4341.25 \quad \text{--- (1)}$$

Applying three moment theorem for B, C & D supports

$$M_B L_1 + 2M_C (L_1 + L_2) + M_D (L_2) = -\frac{6A_1a_1}{L_1} - \frac{6A_2a_2}{L_2}$$

$$14M_B + 2M_C(14+6) + M_D(6) = -3430 - \left(\frac{6 \times \frac{2}{3} \times 6 \times 8 \times 6^2}{8 \times 6} \times 3 \right)$$

$$14M_B + 40M_C + M_D(6) = -3430 - 432 = -3862$$

$$14M_B + 40M_C = -3862 \quad \text{--- (2)}$$

Solving (1) & (2)

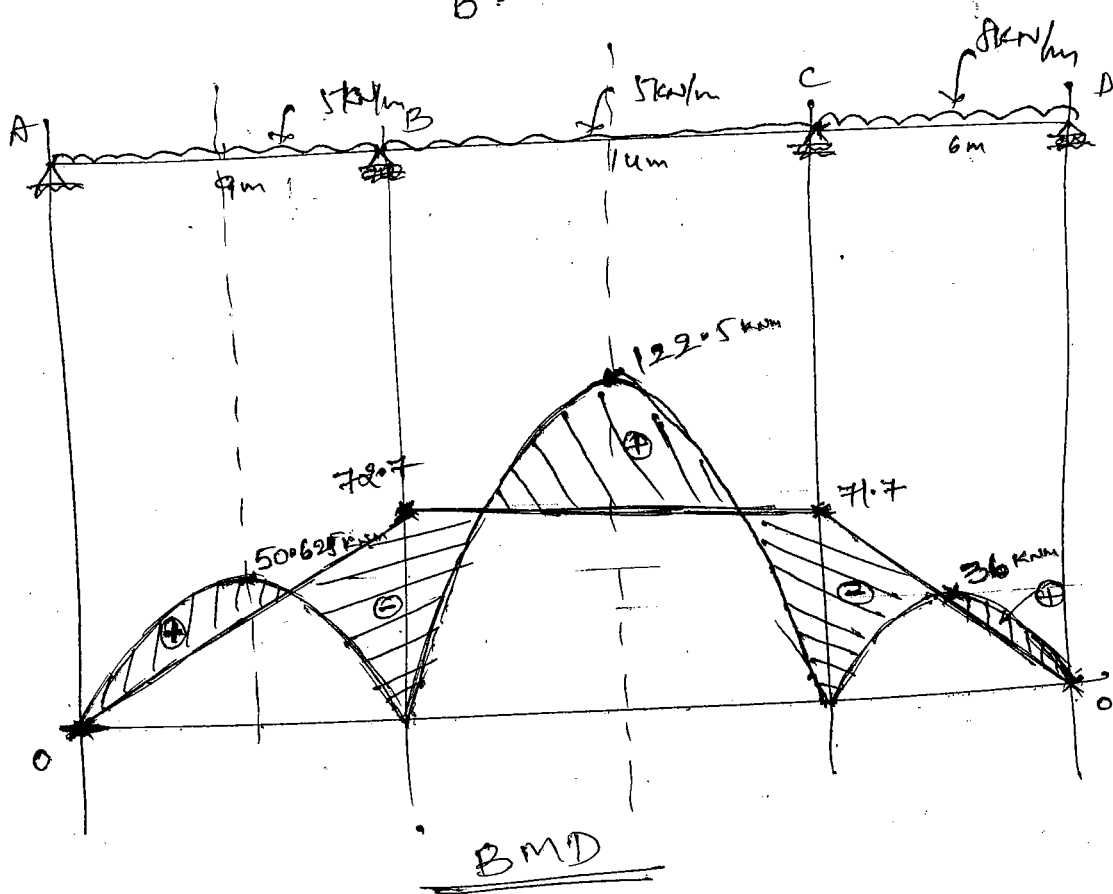
$$-M_B + 0.304 M_C = +94.575$$

$$M_B + 2.857 M_C = -275.852$$

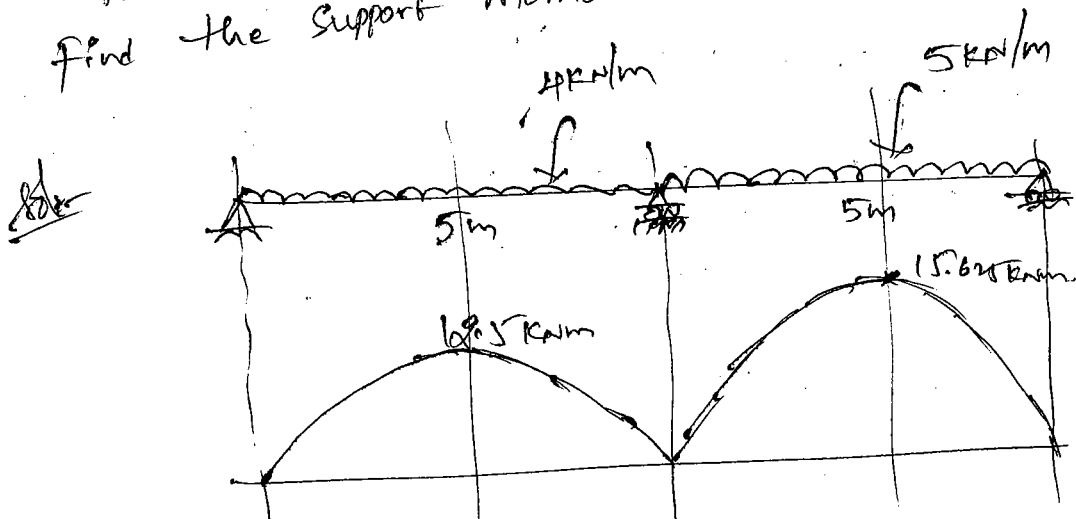
$$2.5526 M_C = -181.482$$

$$\therefore M_C = -71.01 \text{ KNm}$$

$$M_B = -72.7 \text{ KNm}$$



206 Set-2
 prob- A continuous beam ABC of length 10m is simply supported, AB and BC is of 5m length each. Span AB carries an UDL of 4 kN/m and BC carries a UDL of 5 kN/m. Support B sinks down by 5mm below the supports A & C. M.I of beam is 10^8 mm^4 and E is 180 kN/mm^2 . Find the support moments and draw BMD?



Applying Clapeyron's theorem

$$M_A \left(\frac{L_1}{I_1} \right) + 2M_B \left(\frac{L_1}{I_1} + \frac{L_2}{I_2} \right) + M_C \left(\frac{L_2}{I_2} \right) = - \frac{6A_1 a_1}{L_1 I_1} - \frac{6A_2 a_2}{L_2 I_2} + \frac{6Eh_1 a}{L_1} + \frac{6Eh_2 c}{L_2}$$

∴ EI is same throughout,

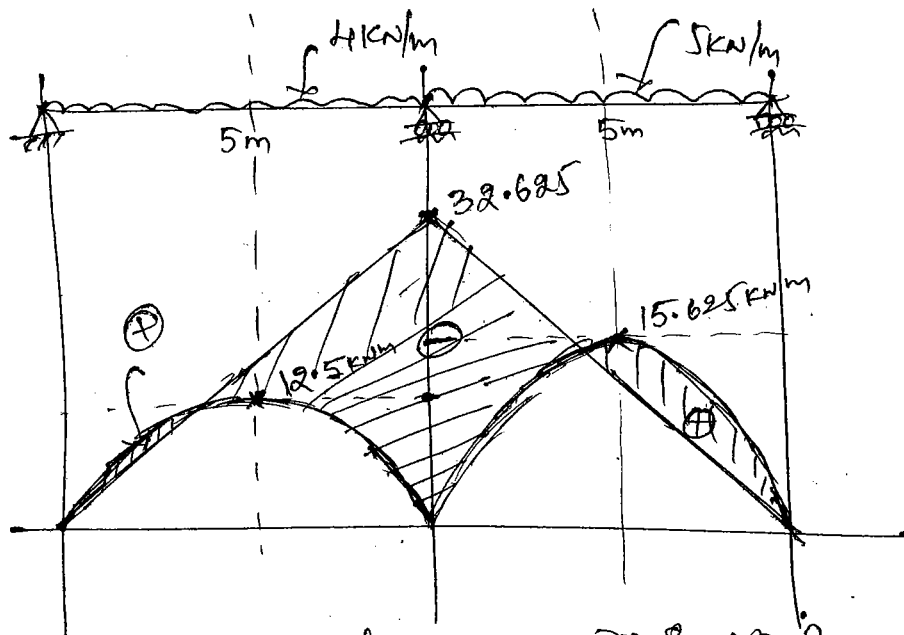
$$M_A \left(\frac{L_1}{I} \right) + 2M_B \left(\frac{L_1 + L_2}{I} \right) + M_C \left(\frac{L_2}{I} \right) = - \frac{6A_1 a_1}{L_1} - \frac{6A_2 a_2}{L_2} + \frac{6Eh_1 a}{L_1} + \frac{6Eh_2 c}{L_2}$$

$$2M_B \times 10 = \frac{-6 \times \frac{8}{3} \times 5 \times 12.5 \times 2.5}{5} - \frac{6 \times \frac{8}{3} \times 5 \times 15.625 \times 2.5}{5} + \frac{6 \times 180 \times 10 \times 10^6 \times \left(\frac{5}{1000} \right)}{5} + \frac{6 \times 180 \times 10^6 \times 10^6 \times \left(\frac{5}{1000} \right)}{5}$$

$$= -125 - 156.25 + 108 + 108$$

$$= -65.25 \Rightarrow M_B = \underline{\underline{-32.625 \text{ kNm}}} = -3.2625 \text{ kNm}$$

BMD:-

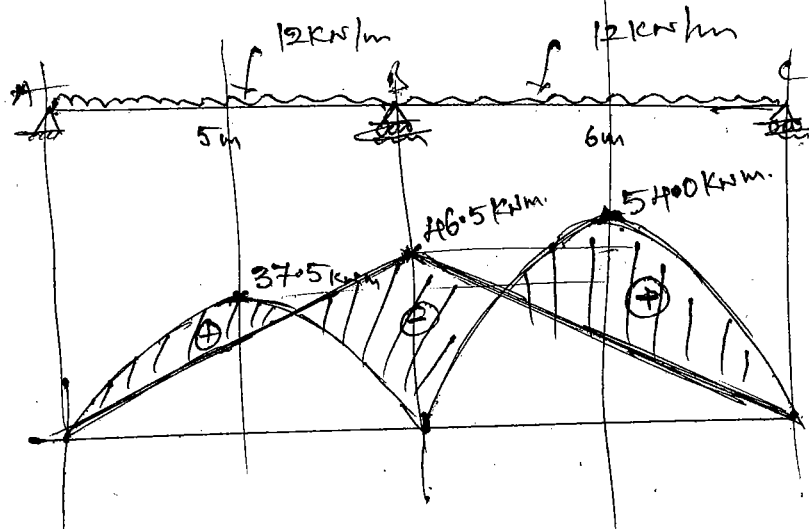


2016
Set-3

Three moment theorem Derivation?

Q4:- proba A continuous beam ABCD is SS over three spans. AB is 8m carrying UDL of 4 kN/m, BC is 8m carrying UDL of 3 kN/m and CD is 5m carrying UDL of 6 kN/m. Find the moments over supports B and C. Draw BMD?

at A and C and continuous over support B with $AB = 5\text{m}$ and $BC = 6\text{m}$. A UDL of 12kN/m is acting over the beam. The moment of Inertia is I throughout the span. Analyse the continuous beam and draw SFD and BMD?



Applying clapeyron's eqⁿ

$$M_A \left(\frac{L_1}{I_1} \right) + 2M_B \left(\frac{L_1}{I_1} + \frac{L_2}{I_2} \right) + M_C \left(\frac{L_2}{I_2} \right) = -\frac{6A_1a_1}{I_1L_1} - \frac{6A_2a_2}{I_2L_2} + \frac{6Eha_A}{L_1} + \frac{6Ehc}{L_2}$$

$\therefore EI$ is same throughout; $h_A = h_C = 0$

$$M_A = M_C = 0 \quad (\text{Simple ends})$$

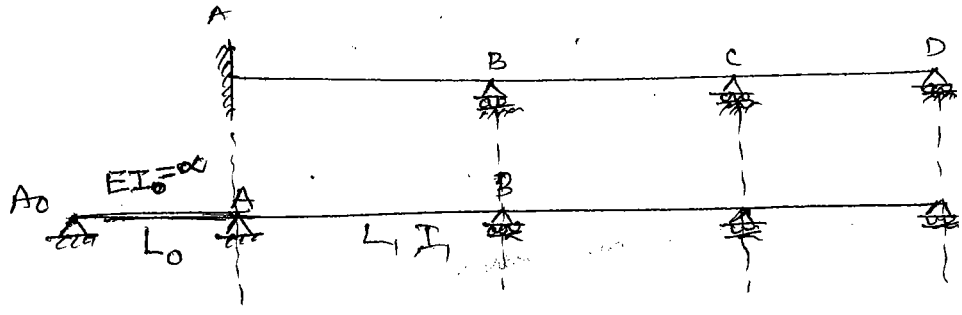
$$\therefore 2M_B(L_1 + L_2) = -\frac{6A_1a_1}{L_1} - \frac{6A_2a_2}{L_2}$$

$$2M_B(5+6) = -6 \times \frac{2}{3} \times \frac{5 \times 37.5 \times 2.5}{5} - 6 \times \frac{2}{3} \times \frac{6 \times 54 \times 3}{6}$$

$$22M_B = -375 - 648 = -1023$$

$$\therefore M_B = 46.5 \text{ kNm}$$

Three moment equation to problems with fixed end :-

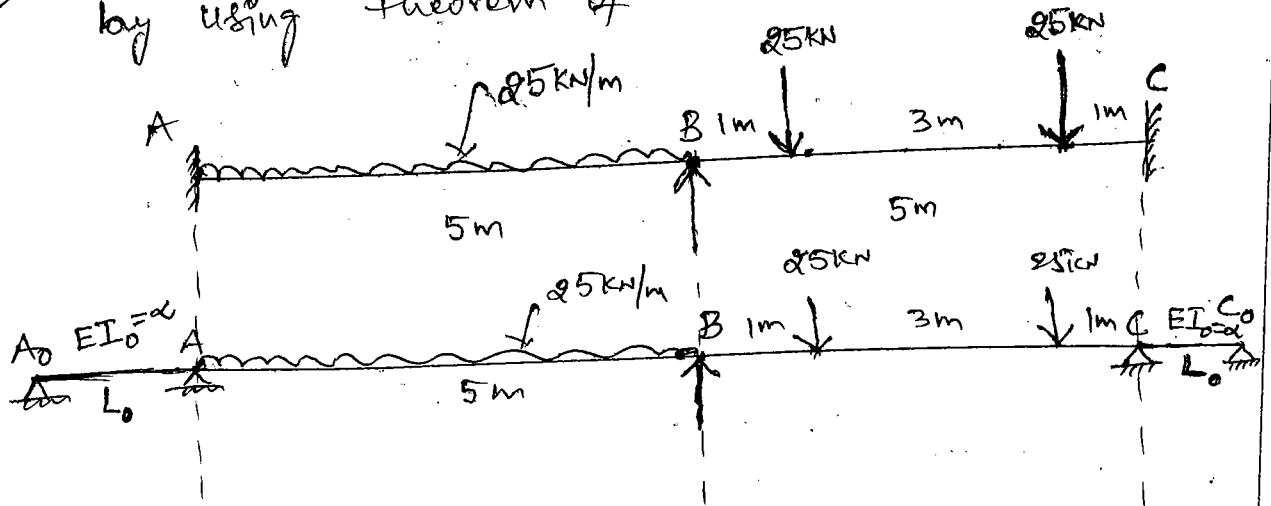


* Add an imaginary span AA0 of length L_0 with infinite flexural rigidity, $EI_0 = \infty$

$$\therefore M_0 \left(\frac{L_0}{I_0} \right) + 2M_A \left(\frac{L_0}{\infty} + \frac{L_1}{I_1} \right) + M_B \left(\frac{L_1}{I_1} \right) = -\frac{6A_0 a_0}{\infty L_0} - \frac{6A_1 a_1}{I_1 L_1} + \frac{6Eh_0}{L_0} + \frac{6Eh_b}{L_1}$$

$$\therefore 2M_A \left(\frac{L_1}{I_1} \right) + M_B \left(\frac{L_1}{I_1} \right) = -\frac{6A_1 a_1}{I_1 L_1} + \frac{6Eh_b}{L_1}$$

Supply prob - Solve the continuous beam in below figure by using theorem of three moments?



Applying the three moment equation for A0, A & B supports

$$\therefore M_0 \left(\frac{L_0}{I_0} \right) + 2M_A \left(\frac{L_0}{I_0} + \frac{L_1}{I_1} \right) + M_B \left(\frac{L_1}{I_1} \right) = -\frac{6A_0 a_0}{I_0 L_0} - \frac{6A_1 a_1}{I_1 L_1} + \frac{6Eh_0}{L_0} + \frac{6Eh_b}{L_1}$$

($\because h_b = 0$)

$$2M_A(L_1) + M_B(L_1) = -\frac{6A_1 a_1}{L_1} \Rightarrow (2M_A + M_B) L_1 = -\frac{6 \times \frac{2}{3} \times 5 \times \left(\frac{25 \times 5}{8} \right) \times 0.5}{L_1}$$

$$2M_A + M_B = -156.25$$

Applying three moment equation to A, B & C supports.

$$M_A \left(\frac{L_1}{I_1} \right) + 2M_B \left(\frac{L_1}{I_1} + \frac{L_2}{I_2} \right) + M_C \left(\frac{L_2}{I_2} \right) = -\frac{6A_1 a_1}{I_1 L_1} - \frac{6A_2 a_2}{I_2 L_2} + \frac{6Eh_a}{L_1} + \frac{6Eh_c}{L_2}$$

$\therefore h_a = h_c$; $I_1 = I_2 = I$

$$M_A(5) + 2M_B(5+5) + M_C(5) = -\frac{6A_1 a_1}{L_1} - \frac{6A_2 a_2}{L_2}$$

$$M_A 5 + 20M_B + 5M_C = -\frac{6 \times \frac{2}{3} \times 5 \times \left(\frac{25 \times 5^2}{8} \right) \times \frac{2.5}{5}}{\frac{2}{3}} - \frac{6 \times 3 \times 3 \times 2.5}{5}$$

$$= -\frac{6 \left[(1 \times 25) + (2 \times 3 \times 3) \right] \times 2.5}{5}$$

$$5M_A + 20M_B + 5M_C = -781.25 - 300 = -1081.25$$

$$M_A + 4M_B + M_C = -216.25 \quad \text{--- (2)}$$

Applying three moment equation to B, C & D supports

$$M_B \left(\frac{L_2}{I_2} \right) + 2M_C \left(\frac{L_2}{I_2} + \frac{L_0}{I_0} \right) + M_D \left(\frac{L_0}{I_0} \right) = -\frac{6A_2 a_2}{I_2 L_2} - \frac{6A_0 a_0}{I_0 L_0} + \frac{6Eh_b}{L_2} + \frac{6Eh_d}{L_0}$$

$\therefore h_b = 0$; $I_2 = I$

$$M_B(5) + 2M_C(5) = -\frac{6A_2 a_2}{L_2} = -300$$

$$M_B + 2M_C = -60 \quad \text{--- (3)}$$

$$2M_A + M_B + 0 = -156.25 \quad \text{--- (1)}$$

$$6M_A + 2M_B + 2M_C = +216.25$$

$$2M_A + 8M_B + 2M_C = -432.5$$

$$6M_B = -216.25 \Rightarrow$$

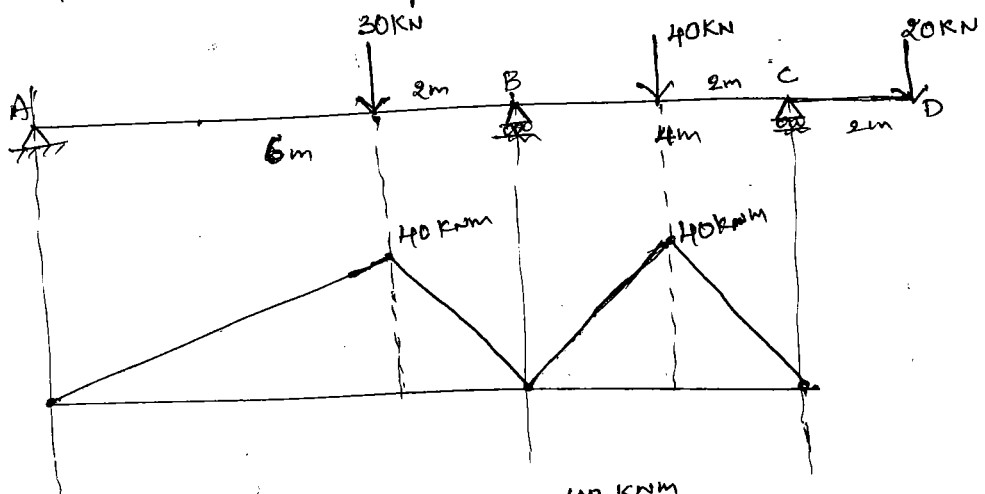
$$M_A = -60.104 \text{ KNm}$$

$$M_B = -36.042 \text{ KNm}$$

$$M_C = -11.98 \text{ KNm}$$

$\therefore M_A = -60 \text{ KNm}$; $M_B = -36 \text{ KNm}$; $M_C = -12 \text{ KNm}$

Prob:- Analyse the continuous beam ABCD shown below, if support C settles down by 5mm. Take $E = 15 \text{ kN/mm}^2$. Moment of Inertia is constant throughout and is equal to $5 \times 10^9 \text{ mm}^4$



$M_A = 0$; $M_C = 20 \times 2 = -40 \text{ kNm}$

Applying three moment Equation,

$$0 + 2M_B(6+4) + M_C L_2 = \frac{-6A_1 a_1}{L_1} - \frac{6A_2 a_2}{L_2} + \frac{6EI h_a}{L_1} + \frac{6EI h_c}{L_2}$$

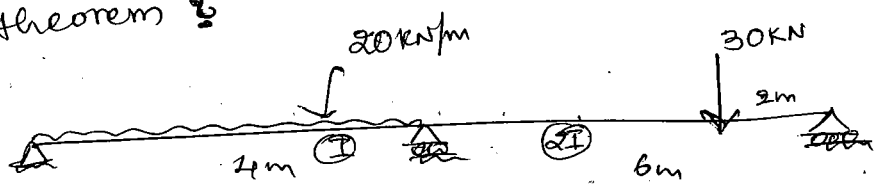
$(h_c = -5)$

$$20M_B - 40 \times 4 = \frac{-6 \times (\frac{1}{2} \times 6 \times 40) \times \frac{(6+4)}{3}}{6} - \frac{6 \times (\frac{1}{2} \times 4 \times 40) \times 2}{4} - \frac{6 \times 15 \times 10^6 \times 5 \times 10^{-3} \times 5 \times 10^{-3}}{4}$$

$M_B = -52.125 \text{ kNm}$

$\therefore M_A = 0$; $M_B = -52.125 \text{ kNm}$, $M_C = -40 \text{ kNm}$

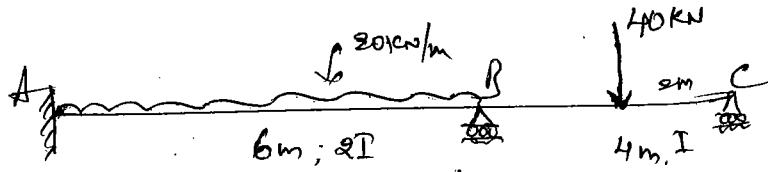
SSBKT- Prob 8- Analyse the two span continuous beam shown below using three moment theorem?



Ans! $M_B = -34.286 \text{ kNm}$

SSCI

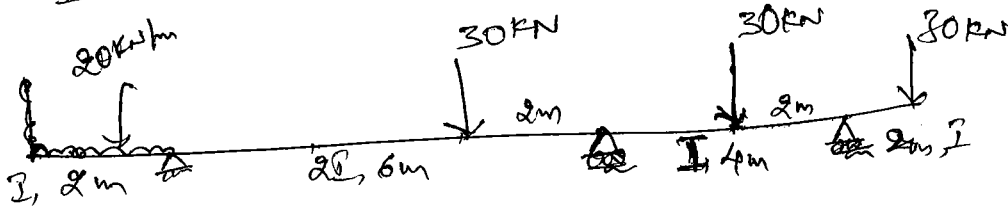
Analyse the continuous beam shown by three moment equation? Draw BMD?



Ans: $M_A = -69.6 \text{ kNm}$; $M_B = -40.8 \text{ kNm}$

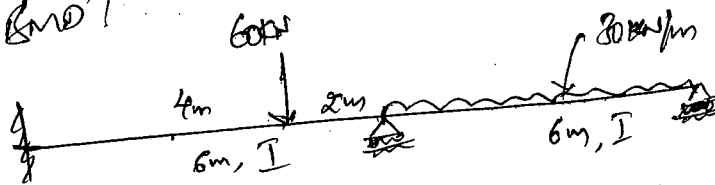
SSCI

Analyse the continuous beam ABCDE shown below & support C sink by 8mm. Given, $E = 200 \text{ kN/mm}^2$
 $I = 0.8 \times 10^8 \text{ mm}^4$? Ans: ($M_C = -1.429 \text{ kNm}$)



SSCI

Analyse the beam ABC shown below and draw BMD?



$M_A = -103.11 \text{ kNm}$
 $M_B = -100 \text{ kNm}$

$\frac{6A_1 a_1}{L} = \left(\frac{8}{60}\right) \omega L^3$

$\frac{6A_2 a_2}{L} = \left(\frac{7}{60}\right) \omega L^3$

$\frac{6A_1 a_1}{L} = \left(\frac{7}{60}\right) \omega L^3$

$\frac{6A_2 a_2}{L} = \left(\frac{8}{60}\right) \omega L^3$

$\left(\frac{5}{32}\right) \omega L^3$

$\left(\frac{5}{32}\right) \omega L^3$

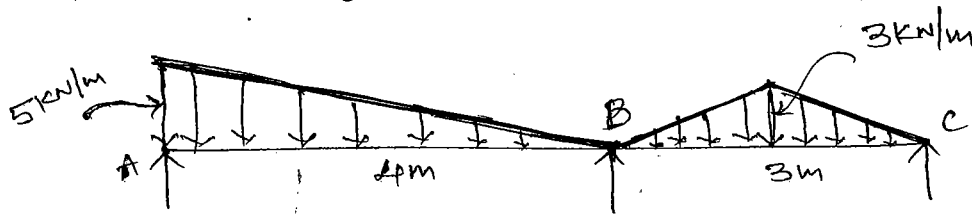
$\frac{\omega}{4L} \left[\frac{a_2^2 (2L^2 - a_2^2)}{a_1^2 (2L^2 - a_1^2)} - \frac{b_2^2 (2L^2 - b_2^2)}{b_1^2 (2L^2 - b_1^2)} \right]$

$-\frac{M}{L} [3(L-a)^2 - L^2] \left(\frac{1}{L}\right) [(3a^2 - L^2)]$

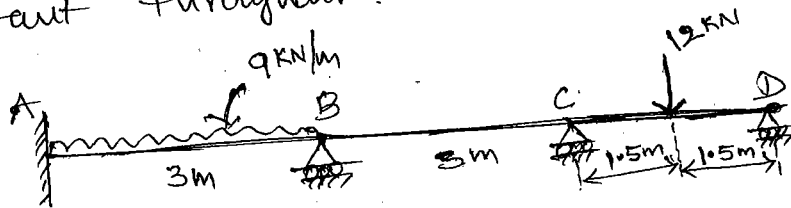
Continuous Beams

Exercise Questions from T.S.M

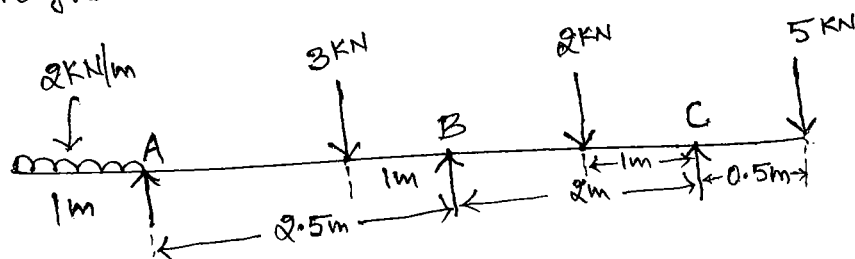
- ① Define a continuous beam?
- ② Derive clapeyron's theorem of three moments including the effect of support settlements?
- ③ Analyse the beam shown below with constant EI using clapeyron's three-moment equation?



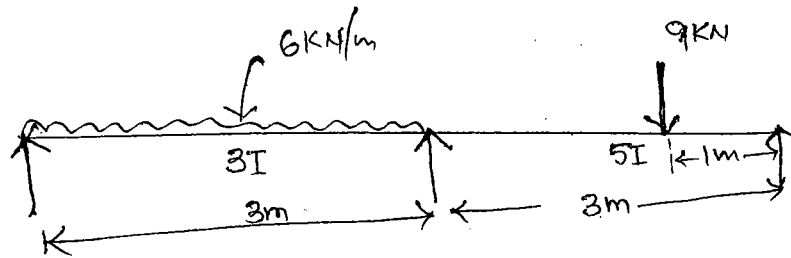
- ④ Using clapeyron's theorem, solve the problem of continuous beam shown below. EI is constant throughout?



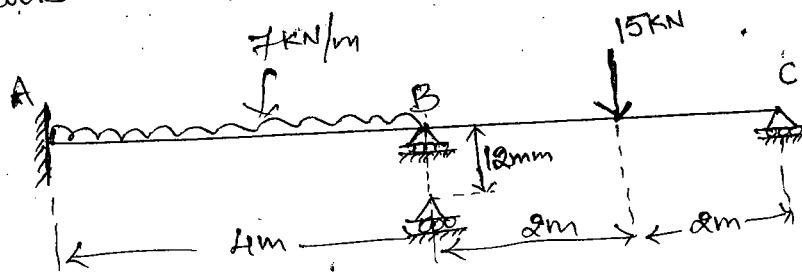
- ⑤ A continuous beam has overhangs on both sides as shown below. Apply three moment equation to determine the support moments. EI is constant throughout



- ⑥ The moment of Inertia of a continuous beam is different for different spans as shown below. Find the reactions?



- ⑦ The support 'B' of a continuous beam shown below has settled by 12mm. Find out the moments at supports?



- ⑧ Draw the SFD and BMD for the continuous beam shown below?

